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1	Towards multiscale simulation of moist flows
2	with soundproof equations
3	Marcin J. Kurowski ^{1,2} , Wojciech W. Grabowski ¹ $*$
4	and Piotr K. Smolarkiewicz ³
5	¹ National Center for Atmospheric Research, Boulder, Colorado, USA
6	² Institute of Meteorology and Water Management, National Research Institute, Warsaw, Poland
7	$^{3}European$ Centre for Medium-Range Weather Forecasts, Reading, UK
R	think active

^{*} Corresponding author address: Wojciech W. Grabowski, NCAR/NESL/MMM, P.O. Box 3000, Boulder, CO 80307-3000. E-mail: grabow@ncar.ucar.edu

ABSTRACT

This paper discusses incorporation of phase changes of the water substance that accompany 9 moist atmospheric flows into the all-scale atmospheric model based on soundproof equa-10 tions. Specific issue concerns developing a theoretical basis and practical implementation to 11 include pressure perturbations associated with atmospheric circulations, from small-scale to 12 global, into representations of moist thermodynamics. In small-scale modeling using sound-13 proof equations, pressure perturbations are obtained from the elliptic pressure solver and are 14 typically excluded from the moist thermodynamics. We argue that in larger-scale flows at 15 least the hydrostatic component of the pressure perturbation needs to be included because 16 pressure variation in synoptic weather systems may affect moist thermodynamics in a way 17 comparable to the temperature variations. As an illustration, we consider two idealized test 18 problems, the small-scale moist thermal rising in a stratified environment and the moist 19 mesoscale flow over an idealized topography. We compare numerical solutions obtained with 20 a fully-compressible acoustic-mode-resolving model and with two versions of the anelastic 21 model, either including or excluding anelastic pressure perturbations in moist thermodynam-22 ics. The two versions of the anelastic model are referred to as the generalized and standard 23 anelastic, respectively. In agreement with the scaling arguments, only negligible differences 24 between anelastic and compressible solutions are simulated. Incorporation of the anelastic 25 pressure perturbations into moist thermodynamics paves the way for future studies where 26 larger-scale moist dynamics will be considered. 27

²⁸ 1. Introduction

Modeling the atmospheric component of the climate system, often referred to as the 29 atmospheric general circulation, has a long history; see collection of reviews in Miller and 30 Smolarkiewicz (2008) and Randall (2010). Because of the computational limitations, early 31 atmospheric general circulation models (AGCMs) featured relatively low spatial resolutions 32 (hundreds to thousands of kilometers) and were based on simplified systems of equations ap-33 propriate for the large-scale atmospheric dynamics. Nevertheless, the critical role of clouds 34 and cloud processes for the Earth energy budget and hydrological cycle has been already ap-35 preciated in early AGCM studies (Arakawa 1975; Charney 1979). Because in early AGCMs 36 clouds were only considered through subgrid-scale parameterizations, their representation 37 was at best questionable. With steadily expanding computational power and increasing 38 awareness of the limitations of cloud parameterizations — the latter sometimes referred 39 to as the cloud parameterization deadlock (Randall et al. 2003) — the past decade wit-40 nessed developments of novel modeling approaches to better represent cloud processes in 41 AGCMs, such as the superparameterization (Grabowski and Smolarkiewicz 1999; Khairout-42 dinov and Randall 2001; Randall et al. 2003) or the convection-permitting nonhydrostatic 43 AGCM (Miura et al. 2007). The emergence of the so-called seamless weather and climate 44 prediction paradigm (Shukla 1998, 2009; Palmer et al. 2008) is another example for appre-45 ciating the small-scale atmospheric phenomena (i.e., weather) in the climate and climate 46 change problem. 47

Reducing the gridlength of AGCMs, down to the convective scale and beyond, is an example of the top-down approach, where increasingly smaller scale processes are resolved rather than parameterized through subgrid-scale modeling. However, one cannot simply reduce the AGCM gridlength to resolve small-scale processes without modifying the model equations to adequately represent the nonhydrostatic dynamics. An alternative approach where nonhydrostatic small-scale models are run applying increasingly large horizontal domains, up to the global scale, can be thought as the bottom-up approach, where cloud-scale processes ⁵⁵ are capable to feed onto progressively larger scales and thus ultimately affect global atmo⁵⁶ spheric dynamics. Experiences of the communities pursuing the top-down or the bottom-up
⁵⁷ approaches accumulate and benefit the quest for the ultimate all-scale AGCM for weather
⁵⁸ and climate research.

It may seem that the bottom-up approach is relatively straightforward as the hydro-59 static dynamics (appropriate for the large-scale flows) may be thought as an asymptotic 60 limit of the nonhydrostatic small-scale dynamics. However, the issue is more complex. 61 Small-scale models available today do not solve generic compressible equations that are 62 valid across the entire range of spatial scales (small-scale turbulence to global). Histor-63 ically, the small-scale atmospheric models evolved along two separate paths and there is 64 significant experience in using such models in weather-related small-scale studies. The first 65 path consists of models built using soundproof equations (e.g., anelastic) with examples 66 of specific models including the NCAR Clark/Hall model (Clark et al. 1996, and refer-67 ences therein), GFDL's Lipps and Hemler model (Lipps and Hemler 1982, 1986), French 68 MesoNH model (http://mesonh.aero.obs-mip.fr/mesonh/) and the EULAG model (Prusa 69 et al. 2008, and references therein). The second path consists of models originating from the 70 compressible dynamics, for instance, the Tapp/White model (Tapp and White 1976) and 71 the Klemp/Wilhelmson model (Klemp and Wilhelmson 1978). NCAR's Weather Research 72 and Forecasting (WRF) Model (http://www.wrf-model.org) is a flagship example of this 73 class of models. However, because compressible dynamics impose severe time-step limita-74 tion due to the presence of fast-propagating sound waves, atmospheric models developed 75 based on compressible dynamics apply various techniques to limit the undesirable impact of 76 physically-irrelevant (for weather and climate) sound waves, e.g., integrating acoustic modes 77 with smaller timestep and low accuracy or applying implicit time integration. Systematic 78 studies of the impact of such techniques on model solutions — for instance, when compared 79 to fully-compressible models — especially when focusing on large-scale dynamics, are rare. 80 We will refer to such models as elastic. 81

The study described in this paper builds upon the interests and recent developments in the area of soundproof systems of equations (e.g., anelastic or pseudo-incompressible). These are briefly reviewed in the next section. A key goal of this work is to progress towards moist thermodynamics valid uniformly in the context of all-scale atmospheric dynamics. Specific aspects that need to be addressed are discussed in section 3. Computational examples illustrating key points of the discussion are presented in sections 4 and 5. A brief summary and outlook in section 6 concludes the paper.

⁸⁹ 2. Modeling atmospheric circulations with soundproof ⁹⁰ equations; an overview

With higher resolution numerical weather prediction (NWP) and climate models becoming available, traditional numerical approaches face new demands. Compressible dynamics is universally valid across the entire range of spatial and temporal scales — from smallscale turbulence to planetary circulations — but impose computational limitations that are difficult to overcome; see e.g. Klein (2011). Consequently, current community efforts focus on alternative, reduced forms of the governing equations for modeling large-scale dry atmospheric motions.

An overall conclusion from the collection of works in Miller and Smolarkiewicz (2008) is 98 that there is no set of governing equations uniformly adopted throughout the NWP com-99 munity, and all operational models differ in some aspects already at the theoretical level. 100 In spite of the ongoing debate on the preferred theoretical formulation of the governing 101 partial differential equations (PDEs), the dominant opinion seems to be that soundproof 102 equations are not appropriate for predicting weather and climate. On the other hand, the 103 soundproof models progress, expand their predictive skill and range of validity, and keep 104 attracting interests of the community. For substantiation, consider an abbreviated list of 105 works exemplifying the community efforts. The list starts with Davies et al. (2003) that 106

quantified departures of normal modes of atmospheric soundproof PDEs from normal modes 107 of the fully-compressible Euler equations. Although the authors questioned the suitability 108 of soundproof equations for weather and climate, their work in fact extended the validity of 109 anelastic models beyond the earlier arguments of scale analyses (Ogura and Phillips 1962: 110 Lipps and Hemler 1982). In Prusa and Smolarkiewicz (2003) and Wedi and Smolarkiewicz 111 (2004) soundproof models were generalized to incorporate time-dependent curvilinear coor-112 dinates, thereby enabling approximations of pliant boundaries — such as finite-amplitude 113 free surface — in soundproof equations and facilitating a coupling of nonhydrostatic anelastic 114 and hydrostatic primitive equation models; see Wedi and Smolarkiewicz (2004) for examples. 115 More recently, Durran (2008) generalized the pseudo-incompressible system (Durran 1989) to 116 spatially inhomogeneous and time-dependent reference states, extending up-scale the accu-117 racy of soundproof approximations. Concomitantly, in Abiodun et al. (2008a) and Abiodun 118 et al. (2008b) the authors compared standard aquaplanet simulations (Neale and Hoskins 119 2000a,b) conducted with three different dynamical cores, including nonhydrostatic anelastic 120 model EULAG (Prusa et al. 2008) within the framework of the Community Atmosphere 121 Model (CAM). They reported favorable comparability of EULAG with the spectral and 122 finite-volume hydrostatic dynamical cores, and found no evidence of inadequacy of anelastic 123 nonhydrostatic equations for climate simulations, epitomized by the aquaplanet benchmark. 124 More recent application of the CAM-EULAG is reported in Abiodum et al. (2011). 125 Arakawa and Konor (2009) proposed a hybrid system of atmospheric PDEs combining non-126 hydrostatic soundproof and hydrostatic primitive equations, thus paving the way for a new 127 class of general circulation models. Using techniques of multiple-scale asymptotic analysis, 128 Klein et al. (2010) showed a formal validity of the Durran pseudo-incompressible and the 129 Lipps-Hemler anelastic equations for realistic magnitudes of the tropospheric potential tem-130 perature stratification, in contrast to single-scale asymptotics of Ogura and Phillips (1962) 131 and common beliefs. Promising results from the application of an EULAG-based anelastic 132 dynamical core to the limited area operational regional NWP model COSMO were reported 133

recently in Ziemianski et al. (2011). On the algorithmic side, Smolarkiewicz and Szmelter
(2009) and Szmelter and Smolarkiewicz (2010a,b) generalized proven conservative numerics
of EULAG to fully unstructured meshes, while sustaining the accuracy of structured-grid
differencing on differential manifolds. This adds yet another path to the advancement of
soundproof models.

Smolarkiewicz (2011) provides a brief discussion of the numerical model EULAG and 139 illustrates the discussion with dry simulations of idealized multiscale flow problems relevant 140 to weather and climate. In addition, a progress towards an unstructured-mesh option of 141 EULAG has been illustrated with simulations of atmospheric wave dynamics across a range 142 of scales. Smolarkiewicz (2011) argues that it is difficult to find a numerical example relevant 143 to NWP and climate studies conclusively showing a failure of soundproof approximations. 144 The cumulative computational experience demonstrated surprising flexibility and a broader 145 than anticipated range of validity of soundproof approximations. Nonhydrostatic soundproof 146 equations imply non-negligible numerical advantages over fully-compressible equations, and 147 the developments of the last decade document growing interest of the community in exploit-148 ing their strengths. 149

3. The moist thermodynamics in the soundproof sys tem

The moist thermodynamics with phase changes of water substance and precipitation (rain and/or ice) impose theoretical and practical challenges that need to be addressed for the all-scale modeling of weather and climate. The extension of the moist thermodynamics to global flows — and flows for which some of the assumptions (e.g., neglecting nonhydrostatic pressure perturbations in moist simulation) may no longer be valid — is the key aspect.

¹⁵⁷ On theoretical grounds, moist thermodynamics involves two issues. The first one is the ¹⁵⁸ latent heating associated with phase changes of the water substance. Latent heating is the

key element of moist processes because it is a source of positive cloud buoyancy in otherwise 159 stably-stratified dry atmosphere. The second issue concerns development and fallout of 160 precipitation. Precipitation comprises a number of effects, from the impact on local buoyancy 161 (through the condensate loading) to the impact of precipitation evaporation outside clouds 162 and precipitation-laden downdrafts that strongly affect properties of the boundary layer. We 163 focus here on the latent heating as its inclusion into the soundproof system brings challenges 164 discussed below. Precipitation processes (ice processes in particular) involve representation 165 of cloud microphysics, but its inclusion in the soundproof system is relatively straightforward. 166 For the ideal gas, the laws of thermodynamics allow introduction of the potential tem-167 perature $\theta = T(p_{oo}/p)^{R_d/c_p}$ as an invariant of adiabatic processes. The potential temperature 168 is also a measure of entropy $s = c_p \ln \theta$, and it can be introduced based on entropy consider-169 ations; see discussions in Bauer (1908), Hauf and Hoeller (1987), Bryan (2008) and Pauluis 170 (2008), among others. An important difference, however, is that entropy considerations in-171 volve assumption of the thermodynamic equilibrium which is only approximately valid for 172 condensation and typically invalid for ice processes. Moreover, applying potential tempera-173 ture as the main thermodynamic variable allows semi-implicit (i.e., implicit with respect to 174 the fast-propagating gravity waves) formulation of the model integration scheme essential 175 for an efficient application of the model in large-scale simulations (Smolarkiewicz et al. 2001; 176 Grabowski and Smolarkiewicz 2002). Based on those arguments and considering that it is 177 conserved in dry adiabatic circulations, we focus on models applying potential temperature 178 θ as the main thermodynamic variable, in addition to mixing ratios of water substance in 179 its various forms; cf. Grabowski (1998, 1999). 180

In the general case of a diabatic flow, the conservation law for the potential temperature includes the heating rate that may include contributions from radiative transfer, chemical reactions, or — the emphasis in our case — phase changes of the water substance. In the latter case, the potential temperature equation becomes: 185

$$\frac{d\theta}{dt} = \frac{L\theta}{c_p T} \frac{dq}{dt} , \qquad (1)$$

where q is the appropriate water mixing ratio, L is the appropriate latent heat, and c_p is 186 the specific heat of air at constant pressure. For the condensation, latent heating is derived 187 through the change of the cloud water mixing ratio. In turn, this change depends on the 188 saturation water vapor mixing ratio $q_{vs} = \epsilon e_s(T)/[p - e_s(T)] \approx \epsilon e_s(T)/p$, where $e_s(T)$ is the 189 saturated water vapor pressure and $\epsilon = R_d/R_v$ (R_d and R_v are the gas constants for the 190 dry air and for the water vapor, respectively). The key point is that changes of both T and 191 p impact q_{vs} . The change of the saturated water mixing ratio Δq_{vs} , due to change of the 192 temperature ΔT and pressure Δp , can be estimated as: 193

$$\Delta q_{vs} = \frac{\partial q_{vs}}{\partial e_s} \frac{de_s}{dT} \Delta T + \frac{\partial q_{vs}}{\partial p} \Delta p \ . \tag{2}$$

However, since the model predicts not the temperature T but the potential temperature θ , one needs locally convert θ into T. Such a conversion involves again pressure p and thus

$$\Delta T = \frac{\partial T}{\partial \theta} \Delta \theta + \frac{\partial T}{\partial p} \Delta p .$$
(3)

¹⁹⁶ Combining (2) and (3) and using the Clausius-Clapeyron relationship for de_s/dT leads to:

$$\frac{\Delta q_{vs}}{q_{vs}} = \beta_L \frac{\Delta \theta}{\theta} - \left(1 + \frac{R}{c_p} \beta_L\right) \frac{\Delta p}{p} , \qquad (4)$$

where $\beta_L = L/R_v T$ is a coefficient that varies between 15 and 20 for air temperatures encountered in the troposphere, and $R/c_p \approx 0.3$. Equation (4) implies that pressure perturbations have approximately an order of magnitude larger impact on q_{vs} through the conversion of θ perturbations into T perturbations [the second term in the parenthesis on rhs of (4)] than the direct effect on q_{vs} (i.e., via the denominator in q_{vs}). In particular, (4) implies that

$$\frac{\Delta q_{vs}}{q_{vs}} \sim 15 \frac{\Delta \theta}{\theta} - 5 \frac{\Delta p}{p} . \tag{5}$$

The first term on rhs of (5) represents effects of the potential temperature change on q_{vs} . In small-scale atmospheric dynamics, the temperature perturbations — or, more generally, perturbations of the density temperature that include effects of water vapor and

cloud condensate on buoyancy (Emanuel 1994) — are a few degrees at most and conse-205 quently $\Delta \theta / \theta \sim 0.01$. Nonhydrostatic pressure perturbations can be estimated using several 206 methods. For instance, the Bernoulli equation implies that the nonhydrostatic pressure per-207 turbation (over the motionless environment) within a rising plume should vary as ρu^2 . This 208 gives pressure perturbations around 1 hPa for the velocity of $\sim 10 \text{ m s}^{-1}$. A similar estimate 209 can be obtained by considering a parcel of a buoyant fluid and estimating pressure perturba-210 tions required to move the air above and below the parcel to allow the parcel to rise. Finally, 211 vortical motions with velocity magnitudes of 10 m s^{-1} also imply pressure perturbations of 212 ~ 1 hPa within the vortex core. Computational example presented in the next section will 213 substantiate these estimates. It follows that $\Delta p/p \sim 0.001$ for typical situations of small-214 scale atmospheric dynamics. Hence, as far as moist thermodynamics is concerned, pressure 215 perturbations can be neglected compared to the potential temperature perturbations as it 216 is typically done in anelastic models; see e.g., Lipps and Hemler (1982); Grabowski and 217 Smolarkiewicz (1996), among many others. 218

However, neglecting pressure perturbations in moist thermodynamics cannot be univer-219 sally valid. For instance, in a tornado, pressure perturbations can reach 100 hPa when 220 the velocities reach 100 m s⁻¹, and then $\Delta p/p \sim 0.1^{1}$. As far as larger-scale dynamics is 221 concerned, temperature and hydrostatic pressure perturbations within midlatitude weather 222 systems can be of the order of 10 K and 10 hPa. Their impact on moist thermodynam-223 ics would then be comparable according to (5). Thus, to extend the validity of the moist 224 soundproof system to small-scale and mesoscale extreme events as well as into larger-scale 225 dynamics (e.g., moist baroclinic waves), one should develop an approach to include pressure 226 perturbations into moist thermodynamics. 227

Traditional thinking is that pressure perturbations obtained from the mass continuity $^{-1}$ Note that we put aside the issue whether the soundproof equations can accurately simulate such a flow in the first place. However, the Mach number squared, an appropriate parameter determining validity of the soundproof system, is ~ 0.1 is such a case. This suggests that the soundproof solution may still be sufficiently accurate.

constraint via the associated elliptic pressure solver in the anelastic or pseudo-incompressible 229 system should not be used in other parts of the model. This is because, for instance, 230 anelastic model dynamics only care about pressure gradients, not the pressure magnitude, 231 and the pressure is only known up to a constant. As will be illustrated by computational 232 examples below, one can design a pressure solver in such a way that the pressure magnitude 233 is predicted in addition to the pressure gradients and thus the pressure can be used in the 234 moist thermodynamics. The key aspect concerns the formulation and application of the 235 boundary conditions and specifics of the generalized Laplacian operator $\sim \nabla \cdot (\mathbf{C} \nabla p)$, where 236 C represents the coefficient matrix; see the appendix for details. 237

For illustration, the next two sections compare idealized two-dimensional moist simu-238 lations performed with the EULAG model applying either anelastic or fully-compressible 239 dynamics options. The anelastic moist EULAG model applies explicit thermodynamics 240 (Grabowski and Smolarkiewicz 2002, and references therein). It is applied in either the 241 original version, that is, with the perturbation pressure excluded from the moist thermo-242 dynamics (Lipps and Hemler 1982; Grabowski and Smolarkiewicz 1996) (referred to as the 243 standard anelastic; ANES) or with the anelastic perturbation pressure combined with the 244 input hydrostatically-balanced pressure profile and used in the moist thermodynamics (re-245 ferred to as the generalized anelastic; ANEG). The compressible model (Smolarkiewicz and 246 Szmelter 2009), referred to as COMP, solves equations of gas dynamics applying nonoscil-247 latory forward-in-time integration scheme (Smolarkiewicz and Margolin 1997), very much 248 alike the anelastic EULAG; cf. section 4 in Smolarkiewicz (2006) for discussion. Extension 249 to moist processes includes replacing dry density by its moist counterpart, with the moist 250 equation of state given by $p = \rho RT(1 + 0.61q_v)$, where ρ is the moist air density and q_v is 251 the water vapor mixing ratio. The compressible model uses the acoustic time step required 252 for numerical stability. This is impractical from the perspective of weather and climate, but 253 leads to accurate compressible solutions, unobscured by numerical devices admitting long 254 time steps in elastic models. Here compressible solutions are considered references for the 255

anelastic models. The emphasis in the discussion will be on the comparison of the perturbation pressure between compressible and anelastic models because these perturbations are used in the moist thermodynamics. Applying the same numerical framework—the anelastic/compressible EULAG model—minimizes the influence of the numerics and better exposes differences in the theoretical model formulation.

4. Anelastic and compressible small-scale dynamics: moist thermals in stratified environment

263 a. Setup of simulations

The computational examples presented in this section address two-dimensional moist 264 thermals rising from rest in a stratified environment, following Grabowski and Clark (1991, 265 1993a,b). A range of initial buoyancy perturbations is considered. The initial buoyancy 266 perturbation combined with water-saturated initial conditions within the thermal leads to 267 the rising motion accompanied by the condensation and latent heat release. As the thermal 268 rises, shear and buoyancy gradients across the cloud-environment interface lead to the devel-269 opment of interfacial instabilities (entraining eddies) as discussed in Grabowski and Clark 270 (1991, 1993a,b). These instabilities are sensitive to details of model numerics and can result 271 in different flow realizations at later times; see, e.g., Figs. 2 and 3 in Grabowski and Smo-272 larkiewicz (2002). To delay the onset of these instabilities, model physics includes explicit 273 diffusion with a constant diffusion coefficient of $2.5 \text{ m}^2 \text{s}^{-1}$. Periodic lateral boundaries and 274 rigid-lid lower and upper boundaries are assumed. As in simulations reported in section 3a of 275 Grabowski and Smolarkiewicz (2002), a uniform 10-m gridlength is used in both horizontal 276 and vertical directions. 277

The environment is assumed to have constant static stability of 1.3×10^{-5} m⁻¹ and relative humidity of 20%, with the temperature and pressure of 283 K and 850 hPa at the lower boundary. The hydrostatically-balanced anelastic profiles of the potential temperature, pressure and density are prescribed as in Clark and Farley (1984, see Eq. 2 therein). The same initial profiles are also used in the compressible model, except for the density profile that is modified to include effects of moisture.

The circular initial temperature perturbation with 300 m radius is imposed in the middle 284 of the $3.8 \times 4 \text{ km}^2$ computational domain. The perturbation center is initially at the height 285 of 800 m above the bottom boundary. The air within the 200 m radius is assumed satu-286 rated and the relative humidity smoothly decreases as in Grabowski and Clark (1991) over 287 100 m towards the 20% environmental relative humidity. The compressible model solves the 288 continuity equation for the moist air density with the pressure derived from the predicted 289 potential temperature and moist air density. Because of that, the initial density perturba-290 tion is derived from the moist equation of state at the onset of the compressible simulation. 291 Three different initial potential temperature perturbations are considered: 0.5, 5 and 50 K. 292 The first one may be considered typical for small convective clouds. The largest perturbation 293 represents magnitude outside the range typical for atmospheric moist convection and is used 294 here to expose differences between anelastic and compressible systems. 295

Table 1 shows comparison of model timesteps and total integration time for all simu-296 lations. A relative ratio of the anelastic and compressible timesteps for 0.5, 5, and 50 K 297 simulations is 100, 25 and 1, respectively. The difference in total integration time is dom-298 inated by this ratio. Both models use the same timestep in the 50 K case, and the total 299 integration time of the anelastic model is about 10% smaller than of the compressible model. 300 This indicates that solving the elliptic problem for the pressure perturbation carries com-301 parable cost to the advection of one additional field, the air density, in the compressible 302 model. 303

304 b. Results

The large difference in the initial buoyancy perturbation for the three cases has a strong impact on the evolution of moist thermal in terms of the vertical velocity and the condensation rate evolutions. To better compare results with various initial perturbations, a non-dimensional buoyancy time scale T_b is introduced following Sanchez et al. (1989):

$$T_b = r_0 (2gr_0 \theta'/\bar{\theta})^{-1/2}, \tag{6}$$

where $r_0 = 300$ m is the initial radius of thermal, θ' is the initial potential temperature perturbation, $\bar{\theta} = 300$ K, and g = 9.81 m s⁻¹ is the gravity constant. The real time t is normalized by the buoyancy time scale T_b . The buoyancy time scale for the initial potential temperature perturbations of 0.5, 5, and 50 K is 95, 30, and 9.5 s, respectively.

In this paper, results are considered from only the first $10 t/T_b$. This is because interfacial 313 instabilities that develop at the leading edge of the thermal in all three cases begin to have 314 a noticeable impact on the solution roughly at 5 t/T_b , and they dominate the solution after 315 10 t/T_b . As a consequence, gradual disintegration of the single convective structure and a 316 formation of a variety of secondary circulations is simulated. The instabilities develop at 317 scales determined by the characteristics of the leading edge interfacial layer, such as the 318 shear and buoyancy gradient; see Grabowski and Clark (1991) for a detailed discussion. 319 Test simulations with different values of the diffusion coefficient showed that adding explicit 320 diffusion can only delay the development, but is not able to suppress it. 321

As an illustration of the above discussion, we show in Fig. 1 the total water fields from all nine simulations (three model formulations and three magnitudes of the initial perturbation) at the nondimensional time of $10 t/T_b$. For the smallest initial perturbation, the solutions correspond to the dimensional time of about 16 min. The solutions are strongly affected by the explicit diffusion; see a diffused pattern of contours near the bottom of the thermal, especially when compared to the other solutions. Condensed water is only present in the central overshooting part of the initial perturbation — it represents late evolution

of the interfacial instability — and this is where the solutions differ between compressible 329 and anelastic models. Similar conclusions apply to the intermediate initial perturbation. 330 However, since these are for dimensional time of around 5 min, the edges of the thermal 331 remain relatively sharp. Solutions for the strongest initial perturbation correspond to only 332 about 2 min of the thermal rise, and the interfacial instabilities seem to evolve in a sur-333 prisingly similar manner between all three model formulations. The only difference between 334 the anelastic and compressible models is slightly slower rise of the compressible thermal as 335 quantified in more detail in the subsequent analysis. 336

Based on results presented in Fig. 1, we show in Fig. 2 pressure perturbations at the 337 nondimensional time of 5 t/T_b . The figure shows that pressure perturbations are dominated 338 by pairs of counter-rotating vortices, with a reduced pressure within their centers, into which 339 the initially circular buoyancy perturbations evolved. Positive pressure perturbations are 340 found in front of the thermal as the thermal pushes upward through the initially motionless 341 air. For each of the initial buoyancy perturbation, the anelastic solution in the left panel 342 (i.e., ANES) agrees well with the compressible solution in terms of spatial structure and 343 magnitude, the latter reaching minimum of about -6, -70 and -1000 Pa, respectively. The 344 figure shows that the anelastic pressure perturbations compare well with the compressible 345 solutions. The solutions based on the generalized anelastic model (ANEG) are shown in the 346 middle column of Fig. 2. These results are practically identical to those from the standard 347 anelastic model; this is consistent with the discussion in the previous section (cf. Eq. 5). 348

Figure 3 compares pattern of the vertical velocity at 5 t/T_b . Again, the differences are very small, except that the 50-K anelastic solution propagates slightly faster than the compressible one as evident from higher vertical velocities. This agrees with our experience with similar dry simulations where even larger temperature perturbations were considered and consequently larger deviations between compressible and anelastic solutions were simulated. Arguably, this is also consistent with a heuristic argument that larger buoyancy perturbations and thus stronger vertical velocities lead to more significant effects of air compressibility above the thermal that impede the thermal rise.

To further support above statements, time evolutions of the minimum and maximum 357 pressure perturbations within the computational domain are shown in Fig. 4. Although 358 pressure solutions from the compressible model presented in Fig. 2 are smooth, one should 359 keep in mind that sound waves form an integral part of the solution. They are induced mostly 360 by an upward movement of the density perturbation, but also by the continuous latent heat 361 release inside the thermal. Sound waves propagate across the entire domain, reflect from 362 model boundaries, and are responsible for the formation of the pressure perturbation field 363 shown in Fig. 2. Their effect is clearly visible in Fig. 4, where compressible solutions feature 364 oscillations with periods between 10 and 20 sec. The latter roughly agrees with the time 365 needed for the sound waves to propagate across the computational domain. In order to 366 reduce the impact of these waves, the compressible model employed divergence damping 367 technique of Skamarock and Klemp (1992). Solutions without divergence damping were 368 similar to those shown here, but with significantly larger oscillations in Fig. 4 and with small 369 wiggles on the pressure perturbation isolines in Fig. 2. The key point is that compressible 370 solutions tend to oscillate close to smooth anelastic solutions, perhaps with the exception of 371 the solutions for the largest initial perturbation. The two anelastic solutions are practically 372 indistinguishable for all cases considered. 373

Figure 5 shows evolutions of the height of the center of mass (barycenter) of the to-374 tal water mixing ratio q_t and the maximum of the updraft strength. The total water was 375 chosen because it is an invariant of the phase change and marks well the evolution of the 376 initial moisture perturbation. For the smallest initial perturbation – slightly larger than 377 in Grabowski and Clark (1991) where only initial humidity perturbation was considered — 378 thermal approximately reaches the level of neutral buoyancy after about four nondimensional 379 time units (i.e., around 400 s) and remains close to this altitude for the rest of the simulation. 380 Maximum vertical velocities are around a few meters per second. Differences between com-381 pressible and anelastic solutions in the latter part of the simulations merely reflect different 382

flow realizations due to interfacial instabilities as illustrated in Fig. 1. This is supported by a 383 set of supplementary simulations with the anelastic model for a range of model timestep Δt . 384 These simulations show that the differences between anelastic and compressible solutions 385 decrease significantly with the decreasing difference in Δt . For instance, the difference in 386 the q_t barycenter height after 10 t/T_b is 138 m for the 2 s anelastic timestep (cf. Fig. 5), 387 but it decreases to 66 m for 0.2 s and 49 m for 0.02 s; the latter timestep is the same as in 388 the compressible model. Moreover, anelastic simulations applying smaller timesteps result 389 in solutions that loose their horizontal symmetry earlier. This suggests that the cumulative 390 numerical (truncation plus round-off) errors have a considerable impact on a development 391 of small-scale interfacial instabilities and thus on the overall behavior of a rising thermal. 392 Therefore, one should keep in mind that the differences between compressible and anelastic 393 solutions can be of various origins, with details of model numerics playing an important role 394 in the comparison. This point is especially valid for simulations with weak forcing, when the 395 difference between the forcing and the numerical noise is relatively small. 396

Solutions for the intermediate temperature perturbation in Fig. 5 are close to each other, 397 with small differences apparent only in the last two nonodimensional time units (about one 398 minute of the dimensional time) as illustrated in Fig. 1. For the largest initial perturbation, 399 thermals rise about 10% faster in the anelastic model than in the compressible model, in 400 agreement with the previous discussion. Differences in the maximum vertical velocity are 401 initially small, $1-2 \text{ m s}^{-1}$, and reach several m s^{-1} in the second half of the simulation. No 402 significant difference between the two anelastic models are present regardless of the initial 403 perturbation. The magnitude of simulated pressure perturbations (Fig. 4) are consistent 404 with the maxima of the vertical velocity as argued in the previous section (i.e., $\sim 10, \sim 100$, 405 and ~ 1000 Pa for the three initial perturbations). 406

The large perturbation of 50 K is used here to illustrate the limits of the anelastic approach, and only in this case the nonhydrostatic pressure perturbations have some (albeit still small) impact on the saturation adjustment. This is documented in Fig. 6 that shows

the evolution of the maximum of the cloud water mixing ratio. The differences between 410 anelastic and compressible solution for nondimensional times larger than two are consistent 411 with the previous discussion (i.e., weaker updraft in the compressible case). Small differences 412 between the two anelastic models are apparent as well. More interesting, however, are the 413 differences during the initial two nondimensional time units, documented in the insert of the 414 figure. The insert shows that the standard anelastic model underpredicts the amount of the 415 cloud water by about 0.1 g kg^{-1} , whereas the compressible and generalized anelastic model 416 agree quite well. The latter is not true during the initial 0.1 of the nondimensional time, 417 that is, during the initial approximately 1 s of the simulation time. The 1-s time corresponds 418 to a period required by sound waves to propagate across the initial buoyancy perturbation 419 to establish corresponding pressure perturbation in the compressible model. This happens 420 instantaneously in the anelastic model. 421

Anelastic and compressible mesoscale dynamics: moist orographic flow

424 a. Setup of simulations

The second example considered in this study examines moist two-dimensional stratified 425 flow over a bell-shaped mesoscale mountain. Experimental setup follows Grabowski and 426 Smolarkiewicz (1996, GS96 hereinafter; see section 4b therein), with parameters exactly 427 as in GS96 unless otherwise stated. As in simulations of the previous section, the moist 428 thermodynamics is limited to the condensation/evaporation only. The height of the mountain 429 is increased to h = 1000 m compared to GS96 to enhance orographically induced cloud 430 water. The uniform inflow horizontal velocity of $U = 20 \text{ ms}^{-1}$ is assumed to maintain 431 similar to GS96 flow regime in terms of the Froude number $U/Nh \approx 3/2$. The relatively cool 432 surface temperature of 273 K ensures absolute stability of the flow and eliminates potential 433

instability which occurs when the equivalent potential temperature decreases with height in 434 the lower troposphere due to the impact of water vapor (see section 6.7 in Emanuel 1994). 435 All simulations apply the subgrid-scale turbulence scheme based on the the turbulent kinetic 436 energy (Margolin et al. 1999). The domain size is $640 \times 18 \text{ km}^2$ with horizontal and vertical 437 gridlength of 2 km and 250 m, respectively. Anelastic/compressible models are integrated 438 applying 10/0.5 s time step. Simulations are run for 6 hrs, sufficient to reach approximately 439 steady-state solutions. The simulations require 49 and 667 s of the wallclock time when 440 using 32 processors for the anelastic and compressible models, respectively. 441

As in the previous section, solutions from the three models are compared: the standard 442 anelastic (ANES), the generalized anelastic (ANEG) and the fully-compressible (COMP). A 443 significant difference from the rising thermal simulations of the previous section is that the 444 orographic flow is differently affected by the round-off and truncation errors. More specifi-445 cally, in the case of a rising moist thermal, these errors affected development of interfacial 446 instabilities and had a strong impact on the flow at later times. Model results for a stable 447 orographic flow with a steady-state solution (i.e., no convection) are affected predominantly 448 by the model mathematical formulation and to a smaller degree by the model numerics. 449

450 b. Results

Figure 7 shows results for the three models, ANES, ANEG, and COMP, after 6 hours 451 of the simulation. Only central 300 km of the horizontal domain is shown. The solution 452 consists of two vertical wavelengths apparent in the fields of the pressure and potential 453 temperature perturbations and the vertical velocity. Perturbations in the upper part of 454 domain attenuate with height due to the presence of the gravity-wave absorber. The vertical 455 velocity pattern is limited to the region directly above the topography (thus documenting the 456 close-to-hydrostatic flow regime), with the pressure and potential temperature perturbations 457 extending horizontally hundreds of kilometers. The patterns are almost identical regardless 458 of the model considered. The two anelastic models provide almost the same solutions that 459

compare well to the fully-compressible solution. Minor differences (around a single contour 460 interval, i.e., ~ 10 Pa) are simulated for the pressure perturbations in the upper part of 461 the domain and away from the mountain. More detailed comparison of the anelastic and 462 compressible pressure perturbations shows that the differences are mostly located in a narrow 463 zone between 4 and 8 km right above the mountain, with extreme values as large as 20 Pa. 464 This region is also characterized by a slight (0.1-0.2 K) underestimation of the potential 465 temperature perturbation by the anelastic models. Cloud fields for all three simulations 466 are very similar. The latter is consistent with the magnitude of temperature and pressure 467 perturbations, up to 5 K and 2 hPa, that according to (5) imply only a small impact of 468 pressure perturbations on the moist thermodynamics. 469

Table ?? shows the extreme values of the fields from Fig. 7 together with the extrema 470 of the difference fields between the anelastic and compressible solutions. The maxima and 471 minima of various fields differ little, typically 1 or 2%, with the largest differences for the 472 maxima of the potential temperature perturbations (up to 4%). The extrema of the difference 473 fields (ANES-COMP and ANEG-COMP) are significantly larger (in the absolute sense) than 474 the differences in the extrema between ANES, ANEG, and COMP. This implies that the 475 differences come from slightly different spatial patterns of model solutions rather than from 476 under- or overprediction of the extreme values. Overall, the table provides no hint as to 477 whether the generalized anelastic model provides solutions that are closer to the compressible 478 model. For instance, the maximum cloud water mixing ratio is 1.037 g kg^{-1} for ANES 479 and 1.049 gkg^{-1} for ANEG and COMP. But the extrema for other fields in the ANES 480 solution are typically closer to COMP than the ANEG, and the cloud water seems to be 481 an exception. Such an ambiguity is also supported by the extrema of the difference fields, 482 with the extrema of the ANES-COMP smaller for some fields and larger for others than 483 ANEG-COMP. Additional simulations of ANEG and ANES models with the same time step 484 as used in the COMP model has not clarified the situation either. Overall, one can only 485 conclude that the differences between various model formulations are small and difficult to 486

⁴⁸⁷ explain, perhaps in agreement with the scaling suggested by Eq. 5.

Time series of the extreme values of pressure perturbations, the maximum of cloud water 488 mixing ratio and the total mass of the condensed water $(\int \rho q_c \, dx \, dz; \text{ in } \text{kg m}^{-1})$ are shown in 489 Figs. 8 and 9. The series demonstrate how the anelastic and compressible models approach 490 their steady state solutions. Pressure fluctuations in the compressible solution are due to 491 sound waves propagating horizontally across the domain. The amplitude of the oscillations 492 decline with time, and after about 5 hours a solution close to the steady state is reached. 493 The evolution of the maximum cloud water mixing ratio and the total condensed water mass 494 show consistent differences between ANES and ANEG models as well as a better agreement 495 between ANEG and COMP. The comparison between the three solutions at the early stage 496 of simulation is highlighted by enlarged panels (a) and (b) in Fig. 9. These show that the 497 generalized anelastic solution better follows the fully-compressible solution. 498

⁴⁹⁹ 6. Summary and outlook

This paper seeks to advance theoretical methodologies and their efficient implementa-500 tions for very-high-resolution nonhydrostatic simulation of the Earth's atmosphere general 501 circulation with soundproof equations. Compressible dynamics is universally valid across 502 the entire range of spatial and temporal scales (i.e., from small-scale turbulence to planetary 503 circulations), but impose computational limitations that are difficult to overcome. Cur-504 rent community efforts focus on alternative forms of the governing equations for modeling 505 large-scale dry atmospheric motions. Efficient numerical simulation of moist processes in 506 very-high-resolution cloud-resolving general circulation models applying those alternative 507 equations is an uncharted territory. 508

There is a significant experience in modeling moist processes at the opposite limits of the spatial scales involved (i.e., small-scale nonhydrostatic versus large-scale hydrostatic dynamics and thermodynamics). A practical approach suitable for multiscale simulation of

weather and climate that combines experiences from large-scale and small-scale dynamics 512 calls for unification of moist thermodynamics. We focus on an approach that applies potential 513 temperature as the main thermodynamic variable because it is conserved in dry adiabatic 514 motions. Alternative approaches, for instance, based on entropy concepts (i.e., the equivalent 515 potential temperature) are deemed inappropriate because of the underlying thermodynamic 516 equilibrium assumption, only approximately valid for water clouds and invalid for ice-bearing 517 clouds. Moreover, semi-implicit integration of the governing equations (Smolarkiewicz 2011; 518 Grabowski and Smolarkiewicz 2002) favor application of the potential temperature as the 519 main thermodynamic variable. 520

The moist thermodynamics require local values of the temperature and pressure. The 521 pressure field is needed not only in the formula for the saturated water vapor mixing ratio, 522 but — more importantly — for the conversion of the potential temperature into temperature. 523 In the soundproof system of equations, the key question is whether the pressure perturba-524 tions obtained from the elliptic pressure solver can be applied in the moist thermodynamics. 525 In traditional small-scale soundproof models, pressure perturbations are typically neglected 526 in the moist thermodynamics and only the environmental hydrostatic pressure profile is used 527 (Lipps and Hemler 1982). A simple scaling argument shows that such an approach is ap-528 propriate for low-Mach-number small-scale flows. However, for flows with appreciable Mach 529 numbers (e.g., ~ 0.1 or larger) pressure perturbations should be used in the moist thermo-530 dynamics. The same is true for larger-scale flows (e.g., midlatitude weather systems) where 531 nearly-hydrostatic pressure perturbations may affect moist thermodynamics in a manner 532 comparable to the potential temperature perturbations. 533

In support of the scaling argument, we compare model solutions to two idealized smallscale and mesoscale moist atmospheric flow problems (rising moist thermal and moist flow over topography, respectively) obtained with anelastic and fully-compressible flow solvers. The anelastic solver that includes pressure perturbations into moist thermodynamics is referred to as generalized anelastic. In agreement with the scaling argument, we document strong similarities between solutions obtained with the compressible, standard anelastic and generalized anelastic flow solvers. It thus follows that the pressure perturbations derived from the elliptic pressure solver in the soundproof system can be applied in the moist thermodynamics. We plan to further substantiate this conjecture by studying diverse cases of moist atmospheric flows for a range of scales and physical scenarios and compare moist solutions obtained with soundproof and fully-compressible options of the EULAG model. Results of such studies will be reported in forthcoming publications.

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APPENDIX

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Conservativity of elliptic solver

A discrete anelastic elliptic pressure equation, at an arbitrary mesh point \mathbf{x}_i and instant tⁿ, can be compactly written as

$$\frac{1}{\rho_o} \nabla \cdot \rho_o \left(\hat{\mathbf{u}} - \mathbf{C} \nabla \phi \right) = 0 .$$
 (A1)

Here: $\rho_o = \rho_o(z)$ is the density of an anelastic reference state, defined by the hydrostatic 562 balance and a constant nonnegative stratification; the differential operator ∇ is identifiable 563 with the vector of primitive discrete spatial partial derivatives in a model code; and $\hat{\mathbf{u}}$ 564 denotes the explicit counterpart of the velocity vector **u** at t^n . The $\mathbf{C}\nabla\phi$ term specifies 565 the implicit complement of $\hat{\mathbf{u}}$ (such that $\mathbf{u} = \hat{\mathbf{u}} - \mathbf{C}\nabla\phi$), where **C** symbolizes a matrix of 566 known coefficients, generally varying with i and n. The role of the implicit complement is 567 to assure the anelastic mass continuity constraint $\nabla \cdot \rho_o \mathbf{u} = 0$ for all \mathbf{x}_i and t^n . At heart of 568 the complement is the implicit potential $\phi \propto (p - p_e)/\rho_o$ — a density normalized pressure 569 perturbation² with respect to an ambient pressure p_e , often (but not necessarily) coinciding 570 with the anelastic reference pressure $p_o = p_o(z)$. Together with suitable boundary conditions 571 and the integrability condition, (A1) forms the generalized Poisson boundary value problem 572 for ϕ . 573

In anelastic models, natural and most common boundary conditions for ϕ are either periodic or Neumann. While the former are straightforward, the latter deserve a comment. A Dirichlet boundary condition for velocity, $\mathbf{n} \cdot \mathbf{u}|_B^n = \mathbf{n} \cdot \mathbf{u}_e$, imply Neumann condition $\mathbf{n} \cdot \mathbf{C} \nabla \phi|_B = \mathbf{n} \cdot (\hat{\mathbf{u}} - \mathbf{u}_e)|_B^n$ for ϕ ; where subscript $_B$ refers to the boundary points, $\mathbf{u}_e(\mathbf{x}, t)$ is a larger scale ambient flow that satisfies the anelastic mass continuity $\nabla \cdot \rho_o \mathbf{u}_e = 0$, and \mathbf{n} is

²The proportionality constant is a model time step or its fraction, depending on the details of numerics.

the outward unit normal to the boundary $\partial \Omega$ of the integration domain Ω . Such a design of the boundary conditions assures the integrability condition $\int_{\partial \Omega} \rho_o \mathbf{n} \cdot \mathbf{u}^n = 0$ for (A1), given $\int_{\partial \Omega} \rho_o \mathbf{n} \cdot \mathbf{u}_e = 0.$

In EULAG, the elliptic problem (A1) is effectively solved to a specified physicallymotivated tolerance, $|| r_{\mathbf{u}} ||_{\infty} \equiv || (\delta t/\rho_o) \nabla \cdot (\rho_o \mathbf{u}) ||_{\infty} \leq \varepsilon$, using the preconditioned generalized conjugate residual algorithm (GCR), an iterative nonsymmetric variational Krylov approach, reviewed recently in Smolarkiewicz and Szmelter (2011). Regardless of the complexity and details of the GCR, an archetype iteration for the problem (A1) may be viewed as

$$\phi^{k+1} = \phi^k + b^k r^k , \qquad (A2)$$

where, k = 0, ..., N numbers the iterations while ϕ^k is a shorthand for the *k*th iterate of ϕ_i^n . Furthermore, *r* denotes the residual error, i.e., the actual value of the lhs of (A1) for ϕ^k , whereas b^k is a coefficient (constant at any given *k*) derived variationally by minimizing residual error of subsequent iteration(s). For either Dirichlet or Neumann boundaries, the recurrence relation (A2) implies, respectively,

 $\phi_B^{k+1} = \phi_B^k + b^k r_B^k , \qquad (A3)$

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$$\mathbf{n} \cdot \mathbf{C} \nabla \phi^{k+1}|_B = \mathbf{n} \cdot \mathbf{C} \nabla \phi^k|_B + b^k \mathbf{n} \cdot \mathbf{C} \nabla r^k|_B , \qquad (A4)$$

The recurrences (A3) or (A4) imply that if the boundary conditions were satisfied at 593 the preceding iteration, they will be satisfied at the subsequent iteration, given that the 594 respective boundary conditions for r or $\mathbf{C}\nabla r$ are homogeneous. Thus, to ensure the correct 595 boundary conditions for ϕ throughout the iteration process, it is important to satisfy them 596 from the outset, at the initialization of the iteration loop, and to maintain the equivalent 597 homogeneous boundary conditions while computing directional vectors, residual errors, and 598 solution-error estimates that enter advanced Krylov-subspace solvers; see Smolarkiewicz and 599 Szmelter (2011) for an exposition. 600

In particular, calculating Neumann boundary condition for ϕ^0 from the ambient flow (as discussed above) assures the correct boundary conditions for all ϕ^k , given the corresponding ⁶⁰³ gradient term $\mathbf{C}\nabla(.)$ of the residual error, directional vectors, etc., is set to zero at the ⁶⁰⁴ boundaries. Furthermore, because (A3) amounts to

$$\rho_o \phi^{k+1} = \rho_o \phi^k + b^k \nabla \cdot (\rho_o \mathbf{u}^k) , \qquad (A5)$$

⁶⁰⁵ its discrete volume integral

$$\int_{\Omega} \rho_o \phi^{k+1} = \int_{\Omega} \rho_o \phi^k + b^k \int_{\partial \Omega} \rho_o \mathbf{n} \cdot \mathbf{u}^k$$
(A6)

606 implies

$$\int_{\Omega} \rho \phi^{k+1} = \int_{\Omega} \rho \phi^k , \qquad (A7)$$

because the constructed boundary condition for velocity assures zero total flux through the model boundary $\partial\Omega$. Thus, initializing the model with $\phi = 0$ at t = 0, assures that the volume integral of ϕ vanishes at all times, thus adding no constant to the solution of the elliptic pressure equation.

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MODEL			EXPE	CRIMENT				
-	$\theta' = 0.5 \ {\rm K}$		θ'	$= 5 { m K}$	heta' :	$\theta' = 50 \ {\rm K}$		
-	Δt [s]	$\begin{array}{c} {\rm run \ time} \\ [{\rm s}] \end{array}$	Δt [s]	run time [s]	Δt [s]	run time [s]		
anelastic compressible	2 0.02	69 4992	$\begin{array}{c} 0.5\\ 0.02 \end{array}$	74 1135	$0.02 \\ 0.02$	369 404		

TABLE 1. Model timesteps Δt and the total integration times (run times) using 32 processors for the anelastic and compressible rising thermal simulations.

TABLE 2. Minima and maxima of the pressure perturbations (p'), vertical velocity(w), cloud water mixing ratio (q_c) and potential temperature perturbations (θ') for the orographic flow solutions obtained with ANES, ANEG, and COMP models after 6 hours. Bottom two rows show minima and maxima of the difference fields ANES-COMP and ANEG-COMP.

	<i>p</i> ′ [Pa]		w $[m s^{-1}]$		(g k	$\frac{q_c}{[gkg^{-1}]}$		θ' [K]	
	min	max	min	max	min	max	\min	max	
ANES	-259.8	91.2	-2.54	2.83	0	1.037	-5.14	5.09	
ANEG	-258.1	91.1	-2.53	2.81	0	1.049	-5.10	5.03	
COMP	-252.4	92.0	-2.62	2.86	0	1.049	-5.17	5.25	
ANES-COMP	-21.2	11.8	-0.15	0.13	-0.07	0.02	-0.32	0.32	
ANEG-COMP	-23.2	12.4	-0.16	0.13	-0.07	0.02	-0.31	0.36	

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FIG. 1. Total water mixing ratio at $10 t/T_d$ for 0.5 K (upper row), 5 K (middle row) and 50 K (bottom row) for ANES (left column), ANEG (middle column) and COMP (right column). Regions of cloud water mixing ratio greater than 0.01 g kg⁻¹ are shaded. Contour interval (CI; in g kg⁻¹) is shown in the upper right corner of each panel.



FIG. 2. Pressure perturbations at 5 t/T_b for 0.5 K (upper row), 5 K (middle row) and 50 K (bottom row) of the initial potential temperature perturbation. Left, middle, and right columns show results for ANES, ANEG, and COMP, respectively. Contour interval (CI; in Pa) is shown in the upper right corner of each panel.



FIG. 3. As in Fig. 2, but for the vertical velocity field. Contour interval (CI; in $m s^{-1}$) is shown in the upper right corner of each panel.



FIG. 4. Evolution of the minimum (left column) and maximum (right column) pressure perturbation for rising thermal simulations with different initial potential temperature perturbation: 0.5 K (upper row), 5 K (middle row), and 50 K (bottom row). ANES and ANEG solutions are almost indistinguishable.



FIG. 5. Evolutions of the height of the total water mixing ratio barycenter (left panel) and the maximum vertical velocity (right panel) for rising thermal simulations with the three different initial potential temperature perturbations.



FIG. 6. Evolution of the cloud water mixing ratio maxima for rising thermal simulations with the initial potential temperature perturbation of 50 K. The insert (a) shows enlarged evolution during the first 1.5 unit of time. See text for a discussion.



FIG. 7. Pressure perturbation (upper row), vertical velocity (2nd row), cloud water mixing ratio (3rd row) and potential temperature (bottom row) at hour 6 of the moist flow over mesoscale topography. Left, middle, and right columns show results for ANES, ANEG, and COMP, respectively. Contour intervals (CI) are shown in the right upper corner of each plot. The dashed line in the cloud water panels represents isoline of 0.01 g kg⁻¹.



FIG. 8. Evolution of the minimum (upper panel) and maximum (lower panel) of the pressure perturbations for ANES, ANEG and COMP orographic flow simulations.



FIG. 9. Evolution of the maximum of the cloud water mixing ratio (upper panel) and total liquid water mass (lower panel) for ANES, ANEG and COMP orographic flow simulations. The inserts (a) and (b) show enlarged evolution during the model spinup. See text for a discussion.